

$$\sum M_z = I_z \omega_z^{\circ} - I_x \omega_y^{\circ} \omega_x + I_y \omega_x^{\circ} \omega_y$$

$$A_y(-1) - B_y(-1) = 0$$

$$A_y = B_y$$

$$\sum F_x \Rightarrow A_x - B_x = 0 \Rightarrow A_x = B_x$$

$$\sum F_y \Rightarrow A_y + B_y - m \cdot g = 0$$

$$A_y + B_y = 2.58(9.81(5.6)) =$$

$$A_y + B_y = 24.525 \text{ N}$$

$$\sum F_z \Rightarrow B_z + A_z = 0$$

$$A_z = -B_z$$

Solve Equations:

$$A_y = B_y, \quad A_x = B_x$$

$$A_y + B_y = 24.525 \text{ N}$$

$$A_z = -B_z$$

$$-A_z + B_z = -506.4775 \text{ Kg} \cdot (\text{rad})^2$$

$$2B_z = -506.4775$$

$$B_z = -253.23875 \text{ Kg} \cdot \frac{1}{2} =$$

$$2A_y = 24.525 \text{ N}$$

$$B_y = A_y = 12.2625 \text{ N}$$

$$A_x = B_x = 0$$

Discussion

Largest load is B_z & $A_z = \pm 253 \text{ N}$

Next is $A_y = B_y = 12 \text{ N}$.

The radial loads generated by the angular

moments are large (~501 kg force), but this does not seem to be too extreme. I need bearings to take this radial load. It seems that's a relatively low load requirement and I imagine head tube bearings could take this, considering when one hits a bump there is a significant lateral load on the head tube bearings.

However, Nixtl did bring up a good point regarding head tube bearings not designed for high rpm. If they are sealed bearings it shouldn't be a problem, and if they are not we could build a bearing block to take the head tube

Further Considerations:

- Need to look @ effects as we speed up design speed as there will be a ω term so slow will mounted on primary axis.
- Affect of Inertia terms due to drive roller

↳ Does not seem to effect bearing loads
But will effect drive requirements as it will be an inertia term we are trying to drive.

The inertia term I used does not account for the mass of the bearings. Look @ the gear's inertia

$$\sum M_y = I_y \omega_y^{\circ} - I_z \omega_x^{\circ} \omega_z + I_x \omega_x^{\circ} \omega_z$$

$$4(-A_z + B_z) = -I_z \omega_x^{\circ} \omega_z$$

$$\Rightarrow 2B_z = \frac{1}{0.4} (I_z \omega_x^{\circ} \omega_z)$$

$$B_z = -1.25 (I_z \omega_x^{\circ} \omega_z)$$

So we are linearly & directly related to each of the terms.

- Loads on primary axle
- Single sided support